



# Tutorial

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**Ex.17:** Using the Bingham plastic and power-law models, determine the various pressure drops, nozzle velocity and nozzle sizes for a section of **12.25** in (**311mm**) hole. Two pumps are used to provide **700** gpm (**2650** l/min). **The Data:**

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- Plastic viscosity = **12** cp,      Yield point = **12** lb/100 ft<sup>2</sup>      Mud weight = **8.8** lb/gal
- Drillpipe (ID = **4.276** in, OD = **5** in, length = **6480** ft)
- Drill collars (ID = **2.875** in, OD = **8** in, length = **620** ft (**189** m))
- Last casing was **13.375** in with an ID of **12.565** in. **13.375** in casing was set at **2550** ft.
- The two pumps are to be operated at a maximum standpipe pressure of **2200** psi.
- Assume a surface equipment type of **4**.

# Power law model:

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## Surface losses

- $\Delta P_s$  or  $P_1 = E \times \rho^{0.8} \times Q^{1.8} \times PV^{0.2}$
- $\Delta P_s$  or  $P_1 = 4.2 \times 10^{-5} \times 8.8^{0.8} \times 700^{1.8} \times 12^{0.2} = \mathbf{52 \text{ psi}}$
- $\theta_{600} = 2 \times PV + YP = 2 \times 12 + 12 = 36$

- $\theta_{300} = PV + YP = 12 + 12 = 24$

- $n = 3.32 \log \frac{\theta_{600}}{\theta_{300}}$

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- $n = 3.32 \log \frac{36}{24} = 0.584$

- $k = \frac{\theta_{300}}{511^n}$

- $k = \frac{24}{511^{0.585}} = 0.628$

## Pipe losses

**Pressure losses inside drill pipe**  $\hat{V} = \frac{24.5 \times Q}{D^2} = \frac{24.5 \times 700}{(4.276)^2} = 937.97 \text{ ft/min}$

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•  $V_c = \left( \frac{5.82 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{1.6 \times (3n+1)}{D \times 4n} \right)^{\left( \frac{n}{2-n} \right)} = \left( \frac{5.82 \times 10^4 \times 0.628}{8.8} \right)^{\left( \frac{1}{2-0.584} \right)} \left( \frac{1.6 \times (3 \times 0.584 + 1)}{4.276 \times 4 \times 0.584} \right)^{\left( \frac{0.584}{2-0.584} \right)}$

= 256 ft/min

- Since  $\hat{V} > V_c$ , flow is turbulent and pressure drop inside drill pipe is calculated from:

•  $P = \frac{8.91 \times 10^{-5} \times \rho^{0.8} \times Q^{1.8} \times PV^{0.2} \times L}{(D)^{4.8}} = \frac{8.91 \times 10^{-5} \times 8.8^{0.8} \times 700^{1.8} \times 12^{0.2} \times 6480}{(4.276)^{4.8}} = 668.45 \text{ psi}$

## Pressure losses inside drill collars

- $\hat{V} = \frac{24.5 \times Q}{D^2} = \frac{24.5 \times 700}{(2.875)^2} = 2074.85 \text{ ft/min}$

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- $V_c = \left( \frac{5.82 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{1.6 \times (3n+1)}{D \times 4n} \right)^{\left( \frac{n}{2-n} \right)} = \left( \frac{5.82 \times 10^4 \times 0.628}{8.8} \right)^{\left( \frac{1}{2-0.584} \right)} \left( \frac{1.6 \times (3 \times 0.584 + 1)}{2.875 \times 4 \times 0.584} \right)^{\left( \frac{0.584}{2-0.584} \right)}$   
 $= 301.8 \text{ ft/min}$

- Since  $\hat{V} > V_c$ , flow is turbulent and pressure loss inside drill collars  $P_3$  is determined from:

- $P_3 = \frac{8.91 \times 10^{-5} \times \rho^{0.8} \times Q^{1.8} \times PV^{0.2} \times L}{(D)^{4.8}} = \frac{8.91 \times 10^{-5} \times 8.8^{0.8} \times 700^{1.8} \times 12^{0.2} \times 620}{(2.875)^{4.8}} = 429.93 \text{ psi}$

## Annular pressure losses /Pressure losses around drill collars

- $$\dot{V} = \frac{24.5 \times Q}{D_{oh}^2 - OD_{dc}^2} = \frac{24.5 \times 700}{12.25^2 - 8^2} = 199.27 \text{ ft/min}$$

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- $$V_c = \left( \frac{3.878 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{2.4 \times (2n+1)}{(D_{oh} - OD_{dc}) \times 3n} \right)^{\left( \frac{n}{2-n} \right)} = \left( \frac{3.878 \times 10^4 \times 0.628}{8.8} \right)^{\left( \frac{1}{2-0.584} \right)} \left( \frac{2.4 \times (2 \times 0.584 + 1)}{(12.25 - 8) \times 3 \times 0.584} \right)^{\left( \frac{0.584}{2-0.584} \right)}$$

= 232.6 ft/min

- Since  $V_c > \dot{V}$ , flow is laminar and pressure loss around drill collars is calculated from:

- $$P_4 = \left( \frac{KL}{300D_e} \right) \left( \frac{2.4\dot{V} \times (3n+1)}{D_e \times 3n} \right)^n = \left( \frac{0.628 \times 620}{300(12.25-8)} \right) \left( \frac{2.4 \times 199 \times (3 \times 0.584 + 1)}{(12.25-8) \times 3 \times 0.584} \right)^{0.584} = \mathbf{6.3 \text{ psi}}$$

## Open hole section- Around Drillpipe

- $$\hat{V} = \frac{24.5 \times Q}{D_{oh}^2 - OD_{dp}^2} = \frac{24.5 \times 700}{12.25^2 - 5^2} = 137.13 \text{ ft/min}$$

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- $$V_c = \left( \frac{3.878 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{2.4 \times (2n+1)}{(D_{oh} - OD_{dp}) \times 3n} \right)^{\left( \frac{n}{2-n} \right)} = \left( \frac{3.878 \times 10^4 \times 0.628}{8.8} \right)^{\left( \frac{1}{2-0.584} \right)} \left( \frac{2.4 \times (2 \times 0.584 + 1)}{(12.25 - 5) \times 3 \times 0.584} \right)^{\left( \frac{0.584}{2-0.584} \right)} = 229.6 \text{ f/min}$$

- Since  $V_c > \hat{V}$ , flow is laminar and the pressure loss around the drillpipe in the open-hole section is determined from:

- $$P_5 = \left( \frac{KL}{300D_e} \right) \left( \frac{2.4\hat{V} \times (3n+1)}{D_e \times 3n} \right)^n = \left( \frac{0.628 \times 3930}{300(12.25-5)} \right) \left( \frac{2.4 \times 137 \times (3 \times 0.584 + 1)}{(12.25-5) \times 3 \times 0.584} \right)^{0.584} = \mathbf{13.76 \text{ psi}}$$

- (Where  $L = 6480 - 2550 = 3930$  ft, and  $L$  = length of drillpipe in the open-hole section).



## Pressure losses around drillpipe: Cased hole section

- $\hat{V} = \frac{24.5 \times Q}{D_c^2 - OD_{dp}^2} = \frac{24.5 \times 700}{12.565^2 - 5^2} = \mathbf{129 \text{ ft/min}}$

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- $V_c = \left( \frac{3.878 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{2.4 \times (2n+1)}{D_e \times 3n} \right)^{\left( \frac{n}{2-n} \right)} = \left( \frac{3.878 \times 10^4 \times K}{\rho} \right)^{\left( \frac{1}{2-n} \right)} \left( \frac{2.4 \times (2n+1)}{(D_c - OD_{dp}) \times 3n} \right)^{\left( \frac{n}{2-n} \right)}$

- $V_c = \left( \frac{3.878 \times 10^4 \times 0.628}{8.8} \right)^{\left( \frac{1}{2-0.584} \right)} \left( \frac{2.4 \times (2 \times 0.584 + 1)}{(12.565 - 5) \times 3 \times 0.584} \right)^{\left( \frac{0.584}{2-0.584} \right)} = \mathbf{183.38 \text{ ft/min}}$

- Since  $V_c > \hat{V}$ , flow is laminar and the pressure loss around the drillpipe in the cased hole is determined from:

- $P_6 = \left( \frac{KL}{300D_e} \right) \left( \frac{2.4\hat{V} \times (3n+1)}{D_e \times 3n} \right)^n = \left( \frac{0.628 \times 2550}{300(12.565-5)} \right) \left( \frac{2.4 \times 129 \times (3 \times 0.584 + 1)}{(12.565-5) \times 3 \times 0.584} \right)^{0.584} = \mathbf{8 \text{ psi}}$

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## Pressure drops across bit

- Total pressure loss in circulating system, except bit.
- $P_{\text{Total}} = P_1 + P_2 + P_3 + P_4 + P_5 + P_6$
- $= 52 + 668.45 + 429.93 + 8 + 13.76 + 6.3 = 1178.44 \text{ psi}$
- Therefore, pressure drop available for bit ( $P_{\text{bit}}$ )
- $P_{\text{bit}} = 2200 - 1164.85 = \mathbf{1021.56 \text{ psi}}$

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## Determine nozzle velocity (ft/s)

- $V_n = 33.36 \sqrt{\frac{P_{\text{bit}}}{\rho}}$
- $V_n = 33.36 \sqrt{\frac{1021.56}{8.8}} = 359.43 \text{ ft/s}$

### Determine total area of nozzles (in<sup>2</sup>)

- $A_n = \frac{0.32*Q}{V_n}$

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- $A_n = \frac{0.32*700}{359.43} = 0.623 \text{ in}^2$

- Nozzle size (in multiples of (1/32))

- $d_n = 32 \sqrt{\frac{4*A_n}{3*\pi}}$

- $d_n = 32 \sqrt{\frac{4*0.623}{3*\pi}} = 16.46$

- Hence, select 1 nozzles of size 17 and 2 of size 16.

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### **B.H.C.P.**

- $B.H.C.P. = 0.052\rho D + \Delta P_{ap} + \Delta P_{ac}$
- $= 0.052*8.8*7100 + 21.76 + 6.3 = \mathbf{3277 \text{ psi}}$
- $E.C.D. = IMW + \frac{\Delta_{ap} + \Delta_{ac}}{0.052 \times D}$
- $E.C.D. = 8.8 + \frac{21.76 + 6.3}{0.052 \times 7100} = \mathbf{8.876 \text{ lb / gal}}$

# CONTACT US

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